

4.35 From the Navier-Stokes equations for incompressible flow in polar coordinates (App. E for cylindrical coordinates), find the most general case of purely circulating motion $v_\theta(r)$, $v_r = v_z = 0$, for flow with no slip between two fixed concentric cylinders, as in Fig. P4.35.

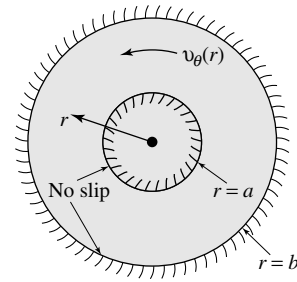


Fig. P4.35

Solution: The preliminary work for this problem is identical to Prob. 4.32 on the previous page. That is, there are two possible solutions for purely circulating motion $v_\theta(r)$, hence

$$v_\theta = C_1 r + \frac{C_2}{r}, \quad \text{subject to } v_\theta(a) = 0 = C_1 a + C_2/a \quad \text{and} \quad v_\theta(b) = 0 = C_1 b + C_2/b$$

This requires $C_1 = C_2 = 0$, or $\mathbf{v}_\theta = \mathbf{0}$ (no steady motion possible between fixed walls) *Ans.*

4.36 A constant-thickness film of viscous liquid flows in laminar motion down a plate inclined at angle θ , as in Fig. P4.36. The velocity profile is

$$u = Cy(2h - y) \quad v = w = 0$$

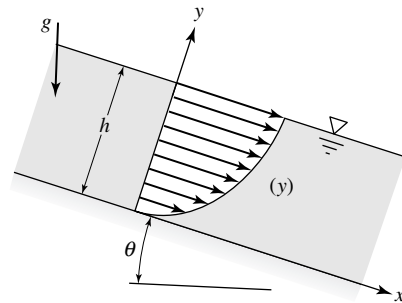


Fig. P4.36

Find the constant C in terms of the specific weight and viscosity and the angle θ . Find the volume flux Q per unit width in terms of these parameters.

Solution: There is atmospheric pressure all along the surface at $y = h$, hence $\partial p / \partial x = 0$. The x -momentum equation can easily be evaluated from the known velocity profile:

$$\rho \left(u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) = -\frac{\partial p}{\partial x} + \rho g_x + \mu \nabla^2 u, \quad \text{or: } 0 = 0 + \rho g \sin \theta + \mu(-2C)$$

$$\text{Solve for } C = \frac{\rho g \sin \theta}{2\mu} \quad \text{Ans. (a)}$$

The flow rate per unit width is found by integrating the velocity profile and using C :

$$Q = \int_0^h u \, dy = \int_0^h Cy(2h - y) \, dy = \frac{2}{3} Ch^3 = \frac{\rho g h^3 \sin \theta}{3\mu} \quad \text{per unit width} \quad \text{Ans. (b)}$$

4.37 A viscous liquid of constant density and viscosity falls due to gravity between two parallel plates a distance $2h$ apart, as in the figure. The flow is fully developed, that is, $w = w(x)$ only. There are no pressure gradients, only gravity. Set up and solve the Navier-Stokes equation for the velocity profile $w(x)$.

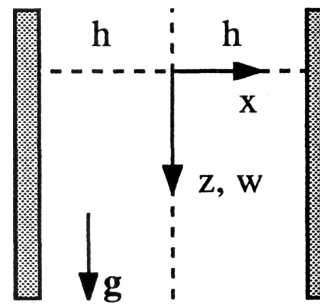


Fig. P4.37

Solution: Only the z -component of Navier-Stokes is relevant:

$$\rho \frac{dw}{dt} = 0 = \rho g + \mu \frac{d^2 w}{dx^2}, \quad \text{or: } w'' = -\frac{\rho g}{\mu}, \quad w(-h) = w(+h) = 0 \quad (\text{no-slip})$$

The solution is very similar to Eqs. (4.142) to (4.143) of the text:

$$w = \frac{\rho g}{2\mu} (h^2 - x^2) \quad \text{Ans.}$$

4.38 Reconsider the angular-momentum balance of Fig. 4.5 by adding a concentrated *body couple* C_z about the z axis [6]. Determine a relation between the body couple and shear stress for equilibrium. What are the proper dimensions for C_z ? (Body couples are important in continuous media with microstructure, such as granular materials.)

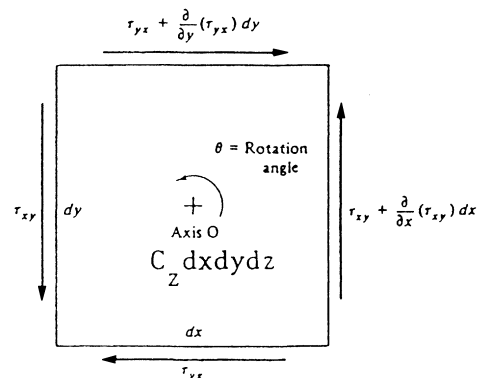


Fig. 4.5

Solution: The couple C_z has to be per unit volume to make physical sense in Eq. (4.39):

$$\left[\tau_{xy} - \tau_{yx} + \frac{1}{2} \frac{\partial \tau_{xy}}{\partial x} dx - \frac{1}{2} \frac{\partial \tau_{yx}}{\partial y} dy \right] dx dy dz + C_z dx dy dz = \frac{1}{12} \rho dx dy dz (dx^2 + dy^2) \frac{d^2 \theta}{dt^2}$$

Reduce to third order terms and cancel $(dx dy dz)$: $\tau_{yx} - \tau_{xy} = C_z$ Ans.

The concentrated couple allows the stress tensor to have unsymmetric shear stress terms.